

Zachary Ahmad

PSE 380

Due 4/19/21

4.7) An underwater turbine extracts a fraction of the kinetic energy from the ocean current. How do the temperature and water velocity influence the power? Hint: write the power term as mass flow rate times specific work.

$$P = \dot{W} = \dot{m} \times w \quad \text{specific work } w = \frac{1}{2} \rho V^2$$

$$\dot{m} = \rho A V \quad \text{so power is } P = \frac{1}{2} \rho A V^3$$

Since temperature of the water enters the turbine with density, and we assume that density remains constant since water is incompressible, power does not vary with temperature.

- I understand how to solve this problem.

①

4.11) A large brewery has a pipe of cross-sectional area 0.2 m^2 flowing carbon dioxide at 400 kPa , 10°C with a volume flow rate of $0.3 \text{ m}^3/\text{s}$. Find the velocity and the mass flow rate.

$$A = 0.2 \text{ m}^2$$

$$P = 400 \text{ kPa}$$

$$T = 10^\circ\text{C} = 283 \text{ K}$$

$$\text{CO}_2, \text{ Table A.5, } R = 0.1889 \text{ kJ/kg}\cdot\text{K}$$

$$\dot{Q} = 0.3 \text{ m}^3/\text{s}$$

assuming ideal gas

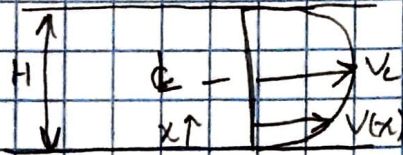
$$\rho = \frac{P}{RT} \rightarrow \frac{400}{0.1889 \cdot 283} = 7.482 \text{ kg/m}^3$$

$$\begin{aligned} \text{Mass flow rate: } \dot{m} &= \rho \dot{Q} \\ &= (7.482)(0.3) \\ &= \underline{2.2446 \text{ kg/s}} \end{aligned}$$

$$\begin{aligned} \text{Volume flow rate: } \dot{Q} &= A \cdot v \\ v &= \frac{\dot{Q}}{A} = \frac{0.3}{0.2} \\ &= \underline{1.5 \text{ m/s}} \end{aligned}$$

• I understand how to solve this problem.

4.15) A flat channel of depth 1 m has a fully developed laminar flow of air at P_0, T_0 with a velocity profile of $V = 4V_c x(H-x)/H^2$, where V_c is the velocity on the centerline and x is the distance across the channel, as shown below. Find the total mass flow rate and the average velocity both as functions of V_c and H .



$$\text{Total Discharge} = Q = \int V dA = \int_0^H V dx = \int_0^H \frac{4V_c x(H-x)}{H} dx = 2V_c(2-H)$$

$$\rho = \frac{P_0}{RT_0}$$

$$\rho Q = \frac{P_0}{RT_0} (2V_c(2-H))$$

$$\text{Average velocity} = Q/A = \frac{Q}{(H \cdot 1)} = \frac{2V_c(2-H)}{H}$$

I do not fully understand how to solve this.

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4.21) A nozzle receives an ideal gas flow with a velocity of 25 m/s, and the exit at 100 kPa, 300 K velocity is 250 m/s. Determine the inlet temperature if the gas is argon, helium, or nitrogen.

$$h_1 + \frac{V_1^2}{2} = h_2 + \frac{V_2^2}{2}, \quad C_p T_1 + \frac{V_1^2}{2} = C_p T_2 + \frac{V_2^2}{2} \quad \text{Table A.5}$$

$$T_1 = T_2 + \frac{1}{C_p} \left(\frac{V_2^2}{2} - \frac{V_1^2}{2} \right) \quad \text{argon, } C_p = 0.52 \text{ kJ/kg K}$$

$$\rightarrow = 300 + \frac{1}{0.52 \times 10^3} \left(\frac{250^2}{2} - \frac{25^2}{2} \right) \\ = \boxed{359.49^\circ\text{C}}$$

$$\text{helium, } C_p = 5.913 \text{ kJ/kg K}$$

$$= 300 + \frac{1}{5.913 \times 10^3} \left(\frac{250^2}{2} - \frac{25^2}{2} \right) \\ = \boxed{305.96^\circ\text{C}}$$

$$\text{nitrogen, } C_p = 1.042 \text{ kJ/kg K}$$

$$= 300 + \frac{1}{1.042 \times 10^3} \left(\frac{250^2}{2} - \frac{25^2}{2} \right) \\ = \boxed{329.70^\circ\text{C}}$$

• I understand how to solve this problem.

④

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4.30) The front of a jet engine acts similarly to a diffuser, receiving air at 900 km/h , -5°C , and 50 kPa , bringing it to 80 m/s relative to the engine before entering the compressor (see below). If the flow area is increased to 120% of the inlet area, find the temperature and pressure in the compressor inlet.

continuity eq: $\dot{m}_i = \dot{m}_e = \frac{AV}{v}$ 

$$\dot{m}_i \left(h_i + \frac{1}{2} V_i^2 \right) = \dot{m}_e \left(\frac{1}{2} V_e^2 + h_e \right), \quad h_e - h_i = c_p (T_e - T_i) = \frac{1}{2} V_i^2 - \frac{1}{2} V_e^2$$
$$= \frac{1}{2} \left(\frac{900 \times 100}{3600} \right)^2 - \frac{1}{2} (80)^2, \quad h_e - h_i = 28050 \text{ J/kg} = 28.05 \text{ kJ/kg}$$

$$\Delta T = \frac{28.05}{1.004} = 27.9, \quad T_e = -5 + 27.9 = \boxed{22.9^\circ\text{C}}$$

$$V_e = V_i \times \frac{1.2 \times 80}{1 \times 250} = V_i \times 0.384$$

$$PV = RT \rightarrow V_e = \frac{RT_e}{P_e} = \frac{RT_i (0.384)}{P_i}$$

$$P_e = P_i \left(\frac{T_e}{T_i} \right) \frac{1}{0.384} = 50 \left(\frac{295.9}{268} \right) \frac{1}{0.384} = \boxed{143.763 \text{ kPa}}$$

• I understand how to solve this problem.

⑤

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4.31) R-410a at -5°C , 700 kPa is throttled, so it becomes cold at -40°C . What is exit P?

Table B.4.1, $h_i = 50.22 \text{ kJ/kg}$ @ -5°C , 700 kPa
 $h_f = 0 \text{ kJ/kg}$, $h_g = 262.83 \text{ kJ/kg}$, $P_{\text{sat}} = 175 \text{ kPa}$
@ -40°C

$$h_e = h_i = 50.22 \text{ kJ/kg} \quad h_f < h_e < h_g$$

so pressure at state 2 is equal to saturation pressure at corresponding temp at state 2.

$$P_e = P_{\text{sat}} = \boxed{175 \text{ kPa}}$$

• I understand how to solve this problem.

①

4.35) Carbon dioxide used as a natural refrigerant flows out of a cooler at 10 MPa, 40°C, after which it is throttled to 1.4 MPa. Find the state (T, x) for the exit flow.

$$T_i = 40^\circ\text{C}, P_i = 10 \text{ MPa}, P_e = 1.4 \text{ MPa}$$

$$h_e + \frac{1}{2}V_e^2 = h_i + \frac{1}{2}V_i^2$$

$$v_e = v_i$$

$$h_i = h_e = 200.14 \text{ kJ/kg}$$

↳ Table B.3.2

@ $P = 1334.2 \text{ kPa}$, $T = -32^\circ\text{C}$, $h_f = 16.3 \text{ kJ/kg}$, $h_{fg} = 307.42 \text{ kJ/kg}$
Table B.3.1 ↗

@ $P = 1427.8 \text{ kPa}$, $T = -30^\circ\text{C}$, $h_f = 20.43 \text{ kJ/kg}$, $h_{fg} = 303.48 \text{ kJ/kg}$

interpolation

$$T_e = -32 + \frac{1400 - 1334.2}{1427.8 - 1334.2} (-30 + 32) = \boxed{-30.6^\circ\text{C}}$$

$$h_f = 16.3 + \frac{1400 - 1334.2}{1427.8 - 1334.2} (20.43 - 16.3) = \boxed{19.20 \text{ kJ/kg}}$$

$$h_{fg} = 307.42 + \frac{1400 - 1334.2}{1427.8 - 1334.2} (303.48 - 307.42) = \boxed{304.65 \text{ kJ/kg}}$$

$$x_e = \frac{h_e - h_f}{h_{fg}} = \frac{200.14 - 19.20}{304.65} = \boxed{0.594}$$

• I understand how to solve this problem.

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4.40) A steam turbine has an inlet of 3 kg/s water at 1200 kPa and 350°C with a velocity of 15 m/s. The exit is at 100 kPa, 150°C and very low velocity. Find the specific work and the power produced.

$$T_i = 350^\circ\text{C} \quad T_e = 150^\circ\text{C} \quad P_i = 1.2 \text{ MPa} \quad P_e = 100 \text{ kPa}$$

$$\dot{m} = 3 \frac{\text{kg}}{\text{s}} \quad V_i = 15 \text{ m/s}$$

$$h_e + \frac{1}{2} V_e^2 + w = h_i + \frac{1}{2} V_i^2$$

$$V_e \approx 0$$

$$w = h_i - h_e + \frac{1}{2} V_i^2$$

Table B.1.3,

@ $P_i = 1200 \text{ kPa}$, $T_i = 350^\circ\text{C}$

$$h_i = 3153.59 \text{ kJ/kg}$$

Table B.1.3

@ $P_e = 100$, $T_e = 150^\circ\text{C}$

$$h_e = 2776.38 \text{ kJ/kg}$$

$$w = 3153.59 - 2776.38 + \frac{1}{2} \frac{(15)^2}{1000} = \boxed{377.3 \text{ kJ/kg}}$$

conversion $\uparrow$

$$P = \dot{W} = \dot{m} w = 3 (377.3) = \boxed{1132 \text{ kW}}$$

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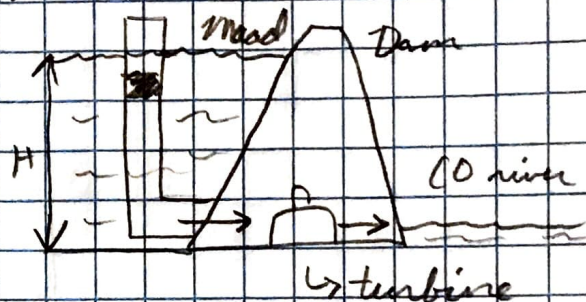
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4.47) Hoover Dam across the Colorado River dams up Lake Mead 200m higher than the river downstream (see below). The electric generators driven by water-powered turbines deliver 1300 MW of power. If the water is 17.5°C, find the minimum amount of water running through the turbines.

continuity: $m_i = m_e$

$$(h + \frac{1}{2}V^2 + gz)_i = (h + \frac{1}{2}V^2 + gz)_e + w_T$$

$$h_i \approx h_e, V_i \approx V_e$$



$$w_T = gz_i - gz_e = 9.81 \times \frac{200}{1000} = 1.961 \text{ kJ/kg}$$

$$\dot{m} = \frac{\dot{W}_T}{w_T} = \frac{1300 \times 10^3 \text{ kW}}{1.961 \text{ kJ/kg}} = 6.63 \times 10^5 \text{ kg/s}$$

$$\dot{V} = \dot{m} v = 6.63 \times 10^5 \times 0.001 = \boxed{664 \text{ m}^3/\text{s}}$$

• I understand how to solve this problem.

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Zachary Almasal

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4.53) A refrigerator uses the natural refrigerant carbon dioxide where the compressor brings 0.02 kg/s from 1 MPa, -20°C to 6 MPa using 2 kW of power. Find the compressor exit temperature.

$$(h + \frac{1}{2} V^2)_e + w = (h + \frac{1}{2} V^2)_i$$

$v_e = v_i = 0$, $w = h_i - h_e$,
steady state

From table B.3.1, for $P_i = 1000 \text{ kPa}$ and $T_i = -20^{\circ}\text{C}$, enthalpy is $h_i = 342.31 \text{ kJ/kg}$

$$h_e = h_i - w = h_i - \frac{\dot{W}}{\dot{m}} = 342.31 - \frac{2}{0.02} = \boxed{442.31 \text{ kJ/kg}}$$

Table B.3.1, @ $P_e = 6000 \text{ kPa}$
and $421.69 \text{ kJ/kg} = h$, $T = 100^{\circ}\text{C}$

B.3.1, @ $P_e = 6000 \text{ kPa}$, $h = 445.02 \text{ kJ/kg}$, $T = 120^{\circ}\text{C}$

$$T_e = 100 + \frac{442.31 - 421.69}{445.02 - 421.69} (120 - 100)$$

$$= \boxed{117.7^{\circ}\text{C}}$$

• I understand how to solve this problem.

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4.62) The air conditioner in a house or car has a cooler that brings atmospheric air from 30°C to 10°C , both states at 101 kPa . For a flow rate of 0.75 kg/s find the rate of heat transfer.

Table A.5 $C_p = 1.004\text{ kJ/kgK}$

$$\dot{Q} = \dot{m}(C_p(T_1 - T_2))$$

$$\dot{Q} = (0.75)(1.004)(30 - 10)$$

$$\dot{Q} = 15.075\text{ kW}$$

• I understand how to solve this problem.

(11)

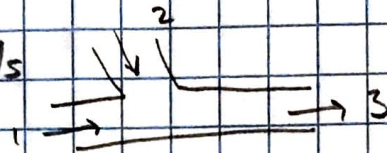
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4.104) Two flows of air are both at 200 kPa; one has 1 kg/s at 400 K and the other has 2 kg/s at 290 K. The two flows are mixed together in an insulated box to produce a single exit flow at 200 kPa. Find the exit temperature.

$$\dot{m}_1 + \dot{m}_2 = \dot{m}_3 = 1 + 2 = 3 \text{ kg/s}$$

$$\dot{m}_1 h_1 + \dot{m}_2 h_2 = \dot{m}_3 h_3$$



$$\dot{m}_1 s_1 + \dot{m}_2 s_2 + \dot{S}_{\text{gen}} = \dot{m}_3 s_3 \rightarrow \text{entropy}$$

$$T_3 = (\dot{m}_1 / \dot{m}_3) T_1 + (\dot{m}_2 / \dot{m}_3) T_2$$
$$= \frac{1}{3} (400) + \frac{2}{3} (290) = \boxed{326.67 \text{ K}}$$

$$\dot{S}_{\text{gen}} = \dot{m}_1 (s_3 - s_1) + \dot{m}_2 (s_3 - s_2)$$

$$= 1 \times 1.004 \ln \left(\frac{326.67}{400} \right) + 2 \times 1.004 \ln \left(\frac{326.67}{290} \right)$$

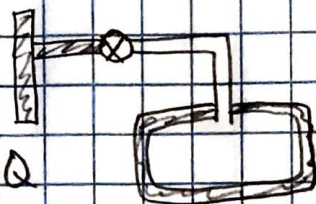
$$= \boxed{0.0358 \text{ kW/K}}$$

• I do not fully understand how to solve this

(12)

4.126) An initially empty cylinder is filled with air from 20°C , 100 kPa until it is full. Assuming no heat transfer, is the final temperature above, equal to, or below 20°C ? Does the final T depend on the size of the cylinder?

$$T_1 = 20^\circ\text{C}, P_1 = 100\text{ kPa}$$



$$m_2 u_2 - 0 = m_1 u_1 - W + Q$$

$$m_2 u_2 = h_1$$

$$m_1 h_1 + 0 - 0$$

$$u_2 = h_1$$

$$\text{so } P_1 = P_2 = 100\text{ kPa}$$

Final temp is above initial temp.

$T_2 > T_1$ so it does not depend on volume.

I understand how to solve this problem.